100% Solar heated house with attic heat store for Ottawa, Ontario

Calculation of heat loss and solar collector area

This is a MathCAD 8 file.

David Delaney, Ottawa, November 21, 2004

The house is a bungalow (one storey) on a well-insulated slab. The slab is raised four or five feet above grade to allow for a tall solar air heater on the south facade of the building. The air heater operates entirely by natural convection, charging a thermal mass situated above it. Since the only exchange of air between the air heater and the rest of the house occurs through the top of the air heater, all air exchange stops when the air heater becomes cold. The thermal mass is located in an overhead attic heat store, about four feet in vertical extent, and covering the whole of the habitable space below. The reminder of the house is assumed to have no thermal mass. The thermal mass in the attic heat store consists of a one foot (0.3 m) thick layer of 1-1/2" to 2-1/2" stones (load: 100 lbf/tt^2, 4800 n/m^2) supported on wire mesh two feet above the floor of the attic heat store. A thermostatically controlled ducted fan draws hot air from the top of the attic heat store to pass through the flow of hot air rising from the air heater so that the cool air may fall through the air heater against the glazing. For details of the thermal scheme of the house see

http://geocities.com/davidmdelaney/thermal-cs/thermal-crawl-space-1.html. For details of the flow organiser see http://geocities.com/davidmdelaney/flow-organiser/flow-organiser.html.

Units:

F≡ <mark>5</mark> K	Btu≡1054 J	C≡K M	$MBtu = 10^6 Btu$	
kJ≡1000 J	kW≡1000 W	$kWh = 3.6 \cdot 10^6 J$	$GJ \equiv 10^9 J$	kBtu≡1000 Btu
$R \equiv 1 \ \frac{ft^2 \cdot hr \cdot F}{Btu}$	$RSI = 1 \cdot \frac{m^2 \cdot K}{W}$	RSI = 5.673•₽	R = 0.1763 • RS	SI

Temperatures:

Th := 70 F Th $- 32 \cdot F = 21.1 \circ C$	Temperature of the habitable space of the house
Tahs := 100 F Tahs - 32 F = $37.8 \circ C$	Average temperature of attic heat store
$Ta := 14 F$ $Ta - 32 F = -10 \circ C$	Ambient (outdoor) air temperature. Used only to model temperature decline when there is no sun
$Tg=50 F$ $Tg-32 F = 10 \circ C$	Temperature of the ground below the house.
Tbase=64.4 F Tbase – $32 \text{ F} = 18 \text{ C}$	Temperature wrt which degree days are measured

Although MathCAD converts automatically between dimensionally consistent units known to it, it cannot do unit conversions that involve addition. So although it converts very well between Fahrenheit and Celsius temperature differences, it cannot convert between temperatures. I have chosen to make Fahrenheit the scale in which temperatures are entered. I enter temperature difference data in whatever scale I obtained the data. I enter degree-days data, for example, in Celsius.

Miscellaneous parameters:

NumberOfPersons≡3	In continuous residence	
baseNonSolarHeatGain := 200 W	Human body heat.	
electricityUsageHeatGain := 400 W	Electric lights, toaster, microwave, computer, TV, radio, CD player, DVD player, VHS, electric stovetop and oven	
nonSolarHeatGain := baseNonSolarHeatGain + electricityUsageHeatGain		
	nonSolarHeatGain = 600 W	
Btu	Thermal capacity of the the attic heat store. 22,000 Btu/F	

thermalCapacity := $22000 \frac{Btu}{F}$ is the thermal capacity of 11 tons water, or 10 tonnes water, or 55 tons of stone, or 50 tonnes of stone water, or 55 tons of stone, or 50 tonnes of stone

thermalCapacity =
$$11.6 \frac{\text{kWh}}{\text{C}}$$



Dimensions:

Lew≡40.ft	Lew = 12.2 m	The east-west length of the interior of the house
LHSns≡30.ft	LHSns = 9.1 m	The north south length of the habitable space
LAHSns≡35·ft	LAHSns = 10.7	m The north south length of the attic heat store
Hahs≡4·ft	Hahs = 1.2 m	The height of the attic heat store
Hhs≡8·ft	Hhs = 2.4 m	The height of the habitable space

R-values:

$Rr \equiv 100 \cdot R$	Rr = 17.6•RSI	Roof
Rwalls≡50·R	Rwalls = 8.8•RSI	Walls of the habitable space
Rf≡20·R	Rf = 3.5•RSI	Floor of habitable space
Rwin≡4 R	Rwin = 0.7•RSI	Windows
Rahs_wall_inc	=7 R Rahs_wall_inc = 1.2 •	RSI attic heat store wall increment

Solar collector (air heater) data:

NumberOfGlazingLayers≡1 Not set up to change this.			
Rah_glazing := NumberOfGlazingLayers·R	R-value of the air heater glazing		
Rah_glazing = 1 •R Rah_glazing = 0.176•RSI	Not currently used		
TransmissionCoefficient≡0.85 ^{NumberOfGlazingLaye}	rs Proportion of incident energy that gets through the glazing.		
TransmissionCoefficient = 0.85			
EnergyRetentionCoefficient=0.6 Guess. Proportion of energy that gets through the glazing that is transferred into the attic heat store.			
airHeaterEfficiency≡TransmissionCoefficient EnergyRetentionCoefficient			
airHeaterEfficiency = 0.51	Proportion of solar radiation energy that falls on the air heater glazing that winds up in the attic heat store.		

Window data:

WindowFractionOfFloor≡0.1

 $W_s = \frac{1}{4} \cdot W_{indow} FractionOfFloor \cdot Lew \cdot LHS_{ns}$ Area of south windows $W_s = 30 \circ ft^2$ $W_s = 2.8 m^2$

Wewn $\equiv \frac{3}{4}$ ·WindowFractionOfFloor·Lew·LHSns Area of east west and north windows

Wewn =
$$90 \circ ft^2$$
 Wewn = 8.4 m²

Conductances:

East west and north windows: $Gwin_ewn \equiv \frac{Wewn}{Rwin}$ $Gwin_ewn = 22.5 \circ \frac{Btu}{hr \cdot F}$ $Gwin_ewn = 11.9 \circ \frac{W}{C}$ South windows $Gwin_s \equiv \frac{Ws}{Gwin_s}$ $Gwin_s = 7.5 \circ \frac{Btu}{Gwin_s}$ $Gwin_s = 4.5 \circ \frac{Ws}{C}$

$$\frac{1}{Rwin} = \frac{1}{Rwin} = \frac{1$$

Attic heat store
$$Gcs = \frac{(2 \cdot Lew + 2 \cdot LAHSns) \cdot Hahs}{Rwalls + Rahs_wall_inc} + \frac{Lew \cdot LAHSns}{Rr}$$

$$Gcs = 24.5 \circ \frac{Btu}{hr \cdot F}$$
 $Gcs = 12.9 \circ \frac{W}{C}$

We treat the floor of the attic heat store as if its conductance is 0, since the sum of all energy flows to the habitable space from the attic heat store is exactly equal to the heat losses to the exterior from the habitable space.

East west and north walls:
$$Gewn_walls = \frac{(Lew + 2 \cdot LHSns) \cdot Hhs - Wewn}{Rwalls}$$

South wall $Gs_wall = \frac{Lew \cdot Hhs - Ws}{Rwalls}$
Floor $Gfloor = \frac{Lew \cdot LHSns}{Rf}$

Ventilation load:

Use a well known crude approximation for the enthalpy change of air: "To raise the temperature of one cubic foot of air by one degree Fahrenheit in one minute requires a heating power of one Btu per hour".

EnthalpyChangeApprox = $\frac{1}{60} \frac{Btu}{ft^3 \cdot F}$

Use ASHRAE 62 outdoor air requirement

ASHRAE_62_requirement_per_person=15 $\frac{\text{ft}^3}{\text{min}}$

NewAirVolumeRate := NumberOfPersons ASHRAE_62_requirement_per_person

NewAirLossPerDegree := EnthalpyChangeApprox · NewAirVolumeRate

NewAirLossPerDegree = $45 \circ \frac{Btu}{hr \cdot F}$ NewAirLossPerDegree = $23.7 \circ \frac{W}{C}$

Degree days analysis:

"EL" = "energy loss"

 $EL_air(days, ddays) := NewAirLossPerDegree \cdot ((Th - Tbase) \cdot days + ddays) \cdot day$

 $EL_air(31, 824 \text{ C}) = 1.9 \cdot 10^9 \text{ oJ}$

 $EL_ewn(days, ddays) := \begin{pmatrix} Gewn_walls ... \\ + Gwin_ewn \end{pmatrix} \cdot ((Th - Tbase) \cdot days + ddays) \cdot day$

 $EL_ewn(31, 824 \text{ C}) = 1.5 \cdot 10^9 \text{ J}$

$$EL_s(days, ddays) := \begin{pmatrix} Gs_wall \dots \\ + Gwin_s \end{pmatrix} \cdot ((Th - Tbase) \cdot days + ddays) \cdot day \cdot \frac{3}{4}$$

South wall only loses energy for 3/4 day $EL_s(31, 824 \text{ C}) = 4.2 \cdot 10^8 \text{ J}$

 $EL_tcs(days, ddays) := Gcs \cdot ((Tahs - Tbase) \cdot days + ddays) \cdot day$

$$EL_tcs(31, 824 \text{ C}) = 1.6 \cdot 10^9 \text{ J}$$

 $EL_floor(days, ddays) := Gfloor \cdot (Th - Tg) \cdot days \cdot day$

EL(days, ddays) := EL_air(days, ddays) ...

EL_floor(31,824 C) = $9.4 \cdot 10^8$ J

+ EL_ewn(days,ddays) ... + $\frac{3}{4}$ ·EL_s(days,ddays) ... + EL_tcs(days,ddays) ... + EL_floor(days,ddays) ... + (-nonSolarHeatGain·24 hr·days)

Months := ("Oct" "Nov" "Dec" "Jan" "Feb" "Mar" "Apr") mnthDays := (31 30 31 31 31 31 30)^T

Solar radiation data (HDdays, irradiancePerDay) from NASA meteorology site, http://eosweb.larc.nasa.gov/sse/, for location 45.316N, 75.666W, (Ottawa, Ontario).

Heating degree days: HDdaysOttawa = $(327 \ 536 \ 824 \ 870 \ 751 \ 619 \ 341)^{T} C$

Parameters for tilted

solar panels: Radiation on equator-pointed tilted irradiancePerDayOttawa := $(2.38 \ 1.87 \ 2.16 \ 2.96 \ 3.54 \ 3.41 \ 2.85)^{T} \frac{kWh}{m^{2}}$ method, Tilt 90 row:

irradiancePerDayOttawa₂ =
$$2.2 \frac{\text{kWh}}{\text{m}^2}$$

$$ELm(mnth) := EL(mnthDays_{mnth}, HDdaysOttawa_{mnth})$$

Oct	$ELm(0) = 2 \circ MMBtu$	$ELm(0) = 2.1 \circ GJ$	ELm(0) = 584 %Wh
Nov	$ELm(1) = 3 \circ MMBtu$	ELm(1) = 3.2 GJ	ELm(1) = 881 %Wh
Dec	ELm(2) = 4.4 • MMBtu	$ELm(2) = 4.7 \circ GJ$	$\text{ELm}(2) = 1.3 \cdot 10^3 \text{ekWh}$
Jan	ELm(3) = 4.7•MMBtu	ELm(3) = 4.9 ° GJ	$ELm(3) = 1.4 \cdot 10^3 \circ kWh$
Feb	ELm(4) = 4.1 •MMBtu	$ELm(4) = 4.3 \circ GJ$	$ELm(4) = 1.19 \cdot 10^3 $ ekWh
Mar	ELm(5) = 3.4•MMBtu	ELm(5) = 3.6 ° GJ	$\text{ELm}(5) = 1 \cdot 10^3 \text{ekWh}$
Apr	ELm(6) = 2.1 • MMBtu	ELm(6) = 2.2 ° GJ	ELm(6) = 600.7 %Wh

Required size of vertical solar aperature:

airHeaterEfficiency = 0.51

SolarArea(mnth) :=	ELm(mnth)
a	$airHeaterEfficiency \cdot mnthDays_{mnth} \cdot irradiancePerDayOttawa_{mnth}$

Oct	SolarArea(0) = 167.1 oft ²	SolarArea(0) = 15.5 sm^2
Nov	SolarArea(1) = 331.5 oft ²	SolarArea(1) = 30.8 m^2
Dec	SolarArea(2) = 409.5 oft^2	SolarArea(2) = 38 m^2
Jan	SolarArea(3) = 314 oft^2	SolarArea(3) = 29.2 m^2
Feb	SolarArea(4) = 229.6 oft^2	SolarArea(4) = 21.3 m^2
Mar	SolarArea(5) = 200.5 oft^2	SolarArea(5) = 18.6 m^2
Apr	SolarArea(6) = 148.3 oft ²	SolarArea(6) = 13.8 m^2

Temperature decline with no sun:

Calculate the temperature of the attic heat store assuming only a small amount of miscellaneous non-solar power input.

 $\operatorname{clock}(t) := t - 24 \cdot \operatorname{floor}\left(\frac{t}{24}\right)$ $\operatorname{TH}(\operatorname{Tahs}) := \operatorname{if}(\operatorname{Th} < \operatorname{Tahs}, \operatorname{Th}, \operatorname{Tahs})$ $\operatorname{Th}(\operatorname{Tahs}) := \operatorname{Th}(\operatorname{Tahs}) := \operatorname{Th}(\operatorname{Ta$

 $\begin{aligned} PwrLossWalls(Tahs, t) &:= (TH(Tahs) - Ta) \cdot \left[(Gewn_walls + Gwin_ewn) \dots + ((clock(t) \ge 9) \cdot (clock(t) < 15) \cdot (Gs_wall + Gwin_s) \right] \end{aligned}$

South wall loses energy only between 9:00 AM and 3:00 PM

 $PwrLossTCS(Tahs) := Gcs \cdot (Tahs - Ta)$

 $PwrLossFloor(Tahs) := Gfloor \cdot (TH(Tahs) - Tg)$

 $PwrAirLoss(Tahs) := (TH(Tahs) - Ta) \cdot NewAirLossPerDegree$

PwrLoss(Tahs, t) := PwrLossWalls(Tahs, t) ... + PwrLossTCS(Tahs) ... + PwrLossFloor(Tahs) ... + PwrAirLoss(Tahs) – nonSolarHeatGain

hours := 1.. 3000

TCS₀ := Tahs Starting temperature for the attic heat store

 $TCS_0 = 100 \circ F$ $TCS_0 - 32 F = 37.8 \circ C$

Difference equation for the temperature of the attic heat store:

$$TCS_{hours} := TCS_{hours-1} - \frac{1}{thermalCapacity} \cdot PwrLoss \left(TCS_{hours-1}, hours-1\right) \cdot 1 hr$$

$$Thh_0 := Th$$
 Starting temperature for the habitable space

Thh₀ Thh₀ =
$$38.9 \circ C$$

$$\text{Thh}_{\text{hours}} := \text{TH}(\text{TCS}_{\text{hours}})$$

tHeatStore :=
$$\frac{\text{TCS}}{\text{F}}$$
 tHabitable := $\frac{\text{Thh}}{\text{F}}$ tAmbient := $\frac{\text{Ta}}{\text{F}}$ freezing := 32

We have calculated with hour ticks for accuracy, but we graph with day ticks.





Simulation of December with fixed solar aperature and grid on:

airHeaterGlazingArea := 40 m² solarPwrDensity₀ := 0 $\frac{W}{m^2}$ Between midnight and 1 AM on december 1

solarPwrDensity_{hours} := if
$$\left[(\operatorname{clock}(\operatorname{hours}) \ge 9) \cdot (\operatorname{clock}(\operatorname{hours}) < 15), \frac{1}{6 \cdot \operatorname{hr}} \cdot \operatorname{irradiancePerDayOttawa_2}, 0, \frac{W}{m^2} \right]$$

 $solar Power Gain_{hours} := air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Glazing Area \cdot air Heater Efficiency \cdot solar Pwr Density_{hours} = air Heater Efficiency \cdot solar Pwr Densit$

$$TAHS_0 := Tahs$$

 $TAHS_0 = 100 \circ F$

$$\begin{aligned} \text{TAHS}_{\text{hours}} &\coloneqq \text{TAHS}_{\text{hours}-1} & \cdots \\ &+ -\frac{1}{\text{thermalCapacity}} \cdot \text{PwrLoss} \left(\text{TAHS}_{\text{hours}-1}, \text{hours} - 1 \right) \cdot 1 \text{ hr} & \cdots \\ &+ \frac{1}{\text{thermalCapacity}} \cdot \text{solarPowerGain}_{\text{hours}-1} \cdot 1 \text{ hr} \end{aligned}$$

$$Thh_0 = 70 \, \text{eV}$$
 $Thh_0 = 38.9 \, \text{eV}$

$$Thh_{hours} := TH(TAHS_{hours})$$

tHeatStore := $\frac{TAHS}{F}$ tHabitable := $\frac{Thh}{F}$ tAmbient := $\frac{Ta}{F}$ freezing := 32

$$tHeatStore_{days} := tHeatStore_{days \cdot 24}$$
 $tHabitable_{days} := tHabitable_{days \cdot 24}$



airHeaterGlazingArea = 40 om^2